

Frequency Response Analysis (FRA)

$$G(s) = \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \quad \text{Second-order transfer function}$$

$$s_{1,2} = -\zeta\omega_n \pm \omega_n\sqrt{\zeta^2 - 1} ; \quad \text{Case } 0 < \zeta < 1$$

K – Gain; ω_n – Natural frequency (rad/sec); ζ – damping ratio

Harmonic Input $x(t) = X\sin\omega t$

X – Amplitude (m) ; ω - frequency (rad/sec)

$$\text{Transfer function } \frac{Y(s)}{X(s)} = G(s)$$

Steady-state response: Output

$$y_{ss}(t) = X|G(j\omega)|\sin(\omega t + \varphi)$$

$$s = j\omega; j = \sqrt{-1};$$

$|G(j\omega)|$ – Magnitude of $(j\omega)$; φ –Phase of $G(j\omega)$

$$|G(j\omega)| = \sqrt{Re^2(\omega) + Im^2(\omega)}$$

$$\varphi = \tan^{-1} \frac{Im(\omega)}{Re(\omega)}$$

$Re(\omega)$ –Real part of $G(j\omega)$; $Im(\omega)$ – Imaginary part of $G(j\omega)$

Example

$$G(s) = \frac{25}{s^2 + 5s + 25} ; K = 1; \omega_n = 5 \text{ rad/sec}; \zeta = 0.5$$

$$G(j\omega) = \frac{25}{(j\omega)^2 + 5j\omega + 25}$$

$$= \frac{1 - 0.04\omega^2}{(1 - 0.04\omega^2)^2 + 0.04\omega^2} - j \frac{0.2\omega}{(1 - 0.04\omega^2)^2 + 0.04\omega^2}$$

Case I Harmonic Input $x(t) = 1 \sin 2t$ m

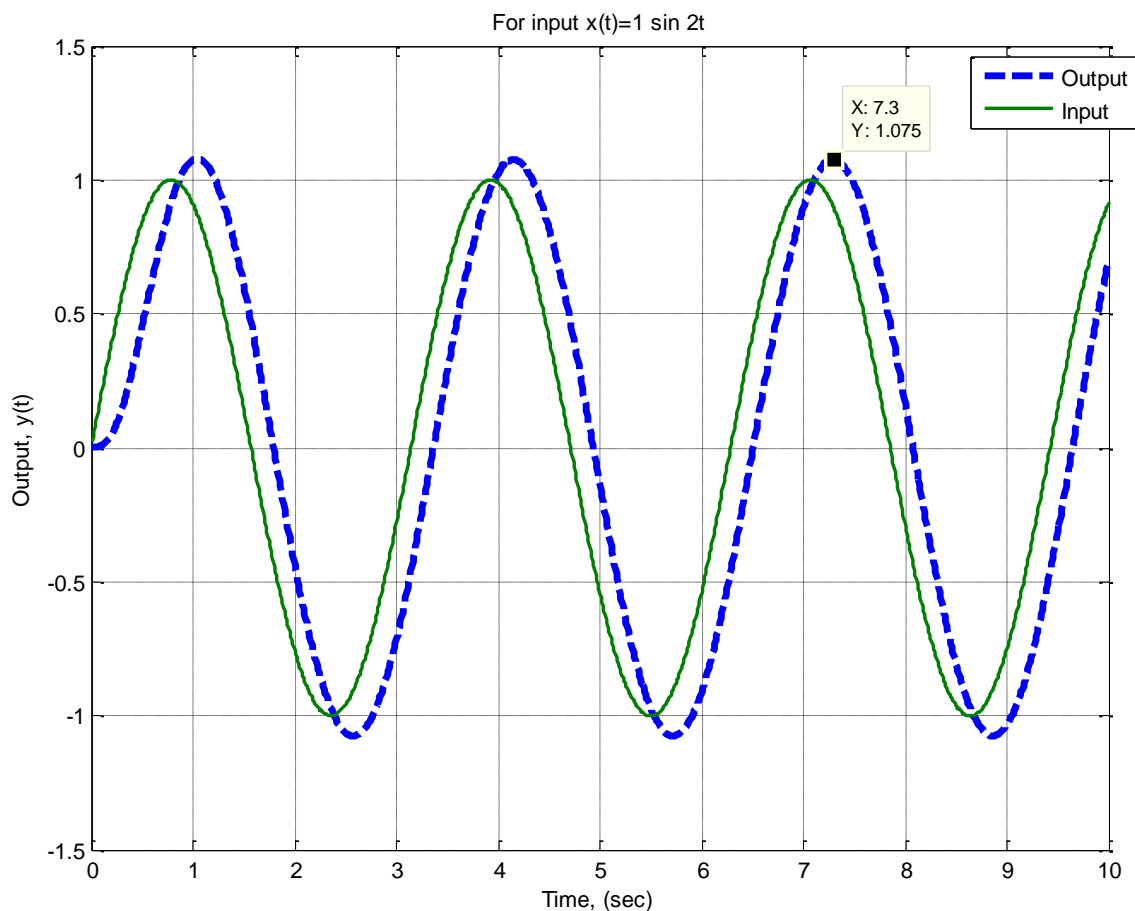
Thus, $Re(\omega) = \frac{0.84}{0.8656} 0.9704$; $Im(\omega) = -\frac{0.4}{0.8656} = -0.462$

$$|G(j\omega)| = \sqrt{(0.9704)^2 + (-0.462)^2} = 1.0747$$

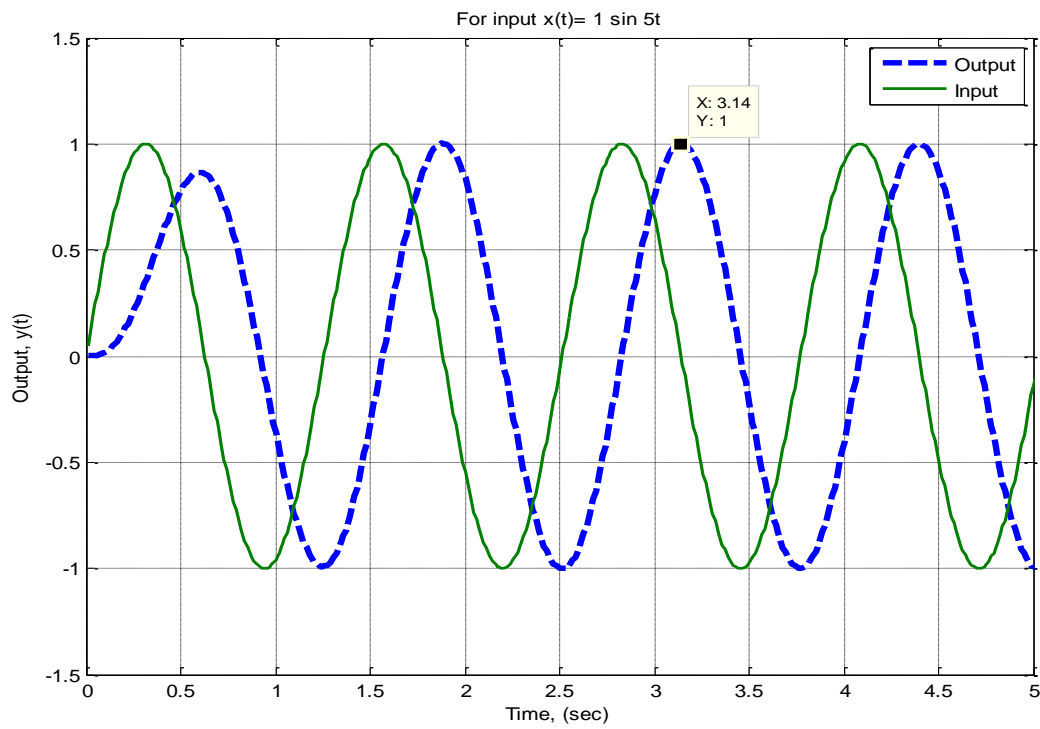
$$\varphi = \tan^{-1} \frac{Im(\omega)}{Re(\omega)} = \tan^{-1} \frac{(-0.462)}{(0.9704)} = -25.45 \text{ degree}$$

Steady-state response: Output

$$y_{ss}(t) = X|G(j\omega)|\sin(\omega t + \varphi) = 1(1.0747)\sin(2t - 25.45^\circ) \text{ m}$$



Case II Harmonic Input $x(t) = 1 \sin 5 t$ m



Case III Harmonic Input $x(t) = 1 \sin 10 t$ m

